



# DESIGN STUDY OF THE MECHANICAL STRUCTURE OF THE SIS300 SUPERCONDUCTING DIPOLES

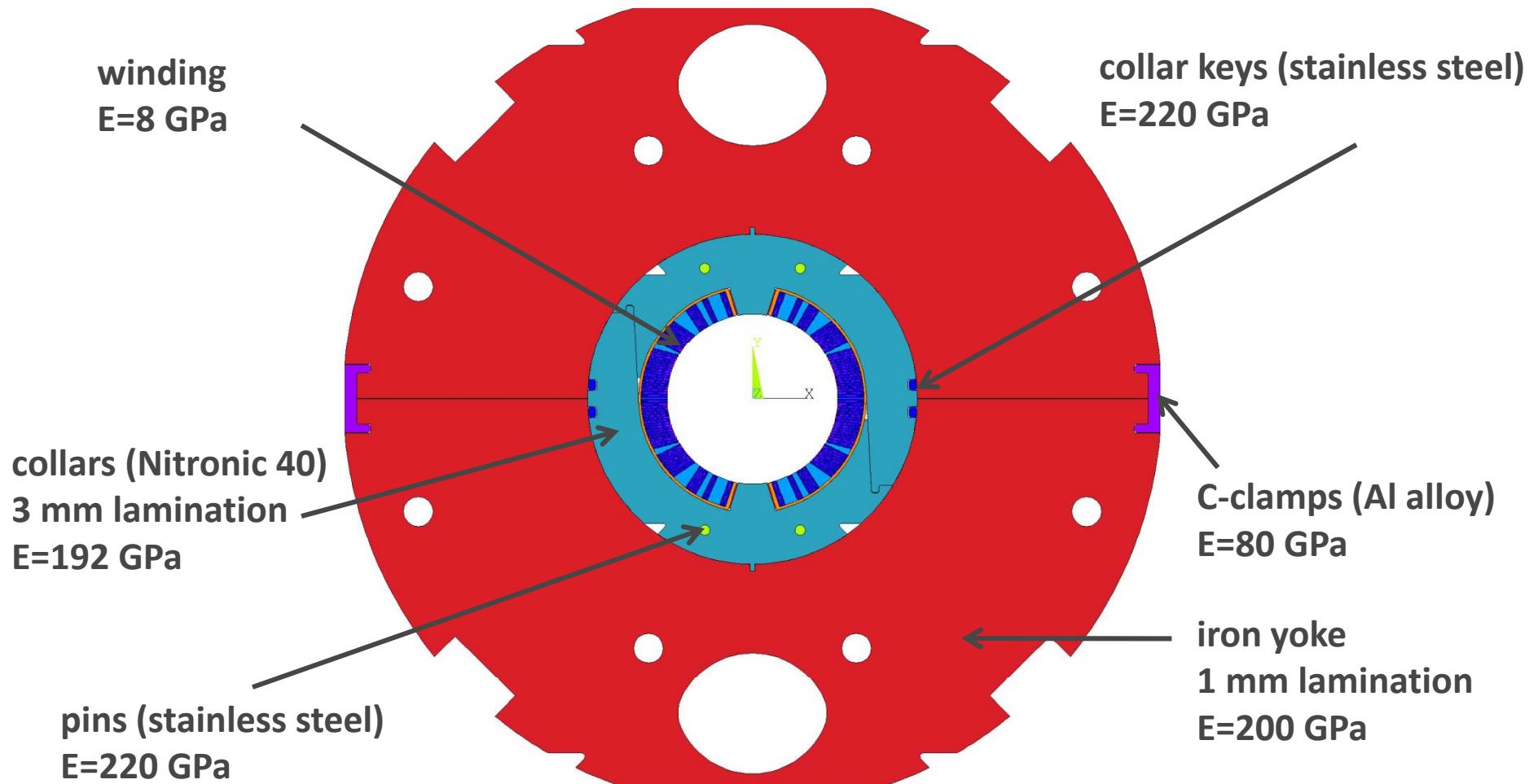
Stefania Farinon  
DISCORAP Review, November 12<sup>th</sup> 2008

# Outline

- 2D mechanical analysis
- Mechanical tests on conductors
- Preliminary 3D mechanical analysis

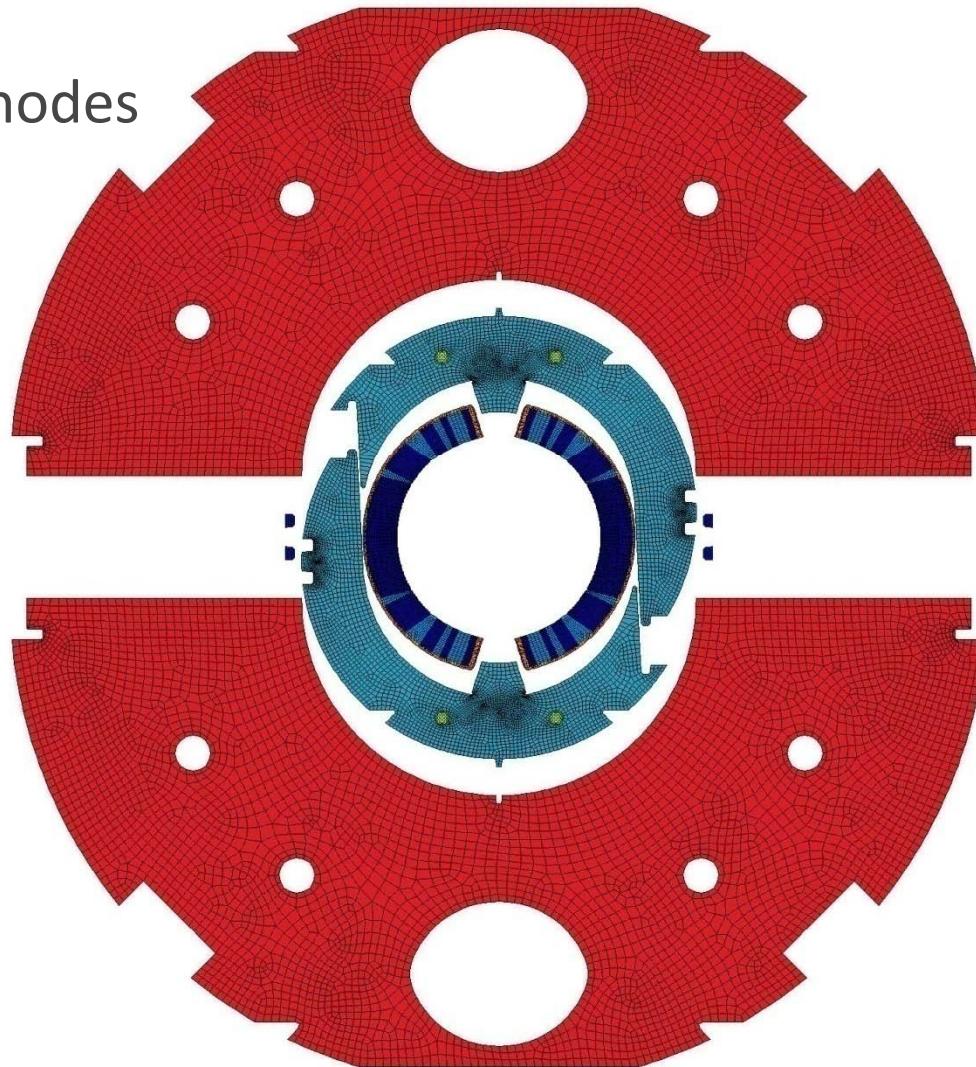
# 2D mechanical analysis

# SIS300 dipole mechanical structure



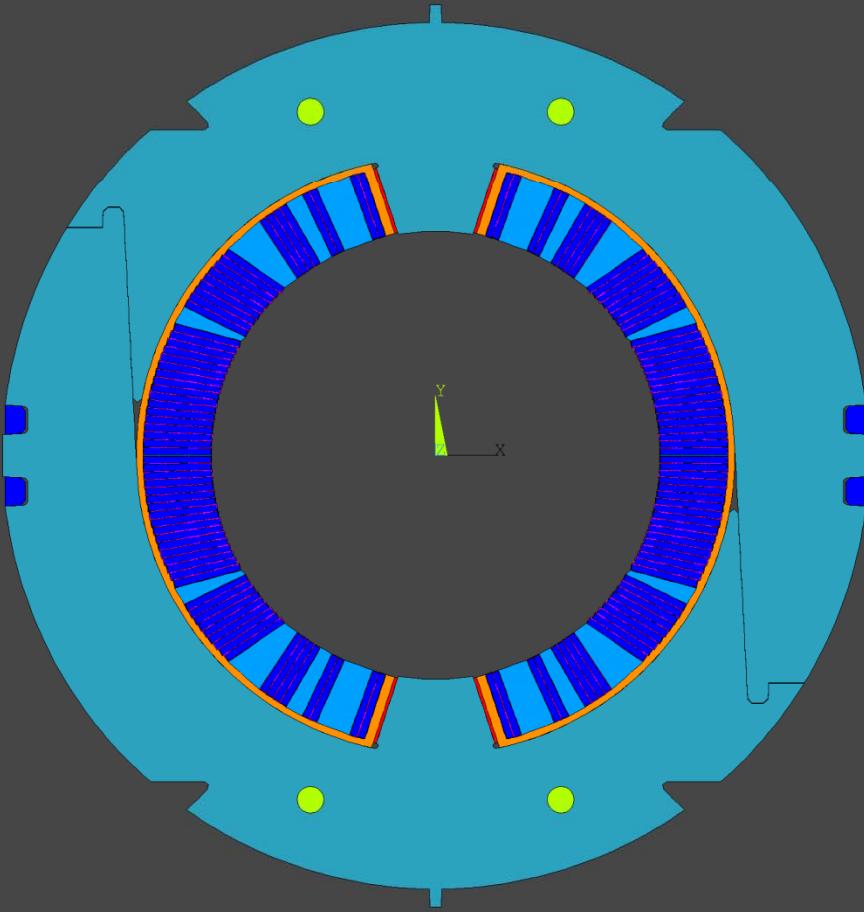
# 2D finite element model

110'500 nodes



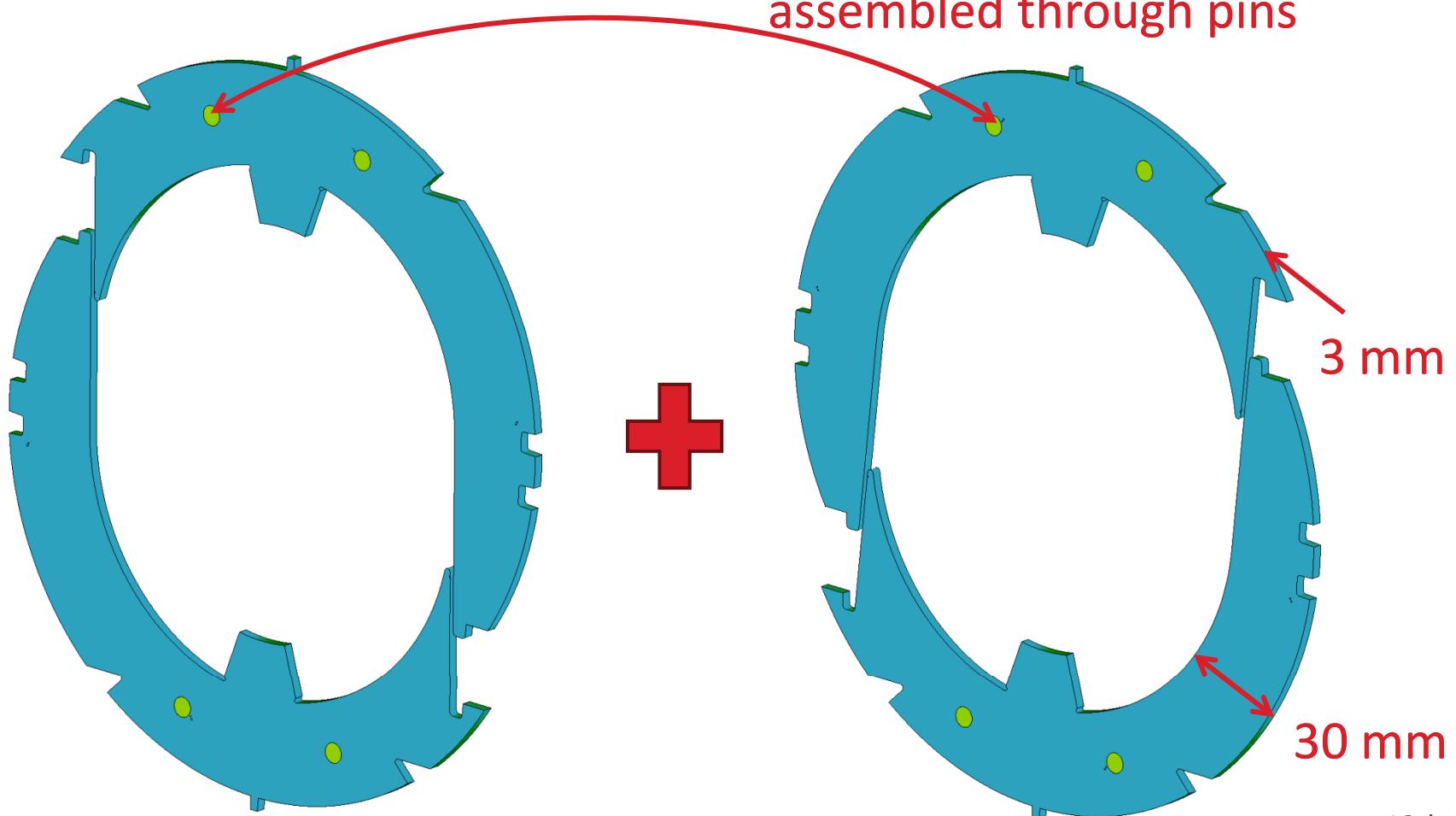
**4 load-steps:**

1. collaring
2. iron yoke assembly
3. cool-down
4. energization

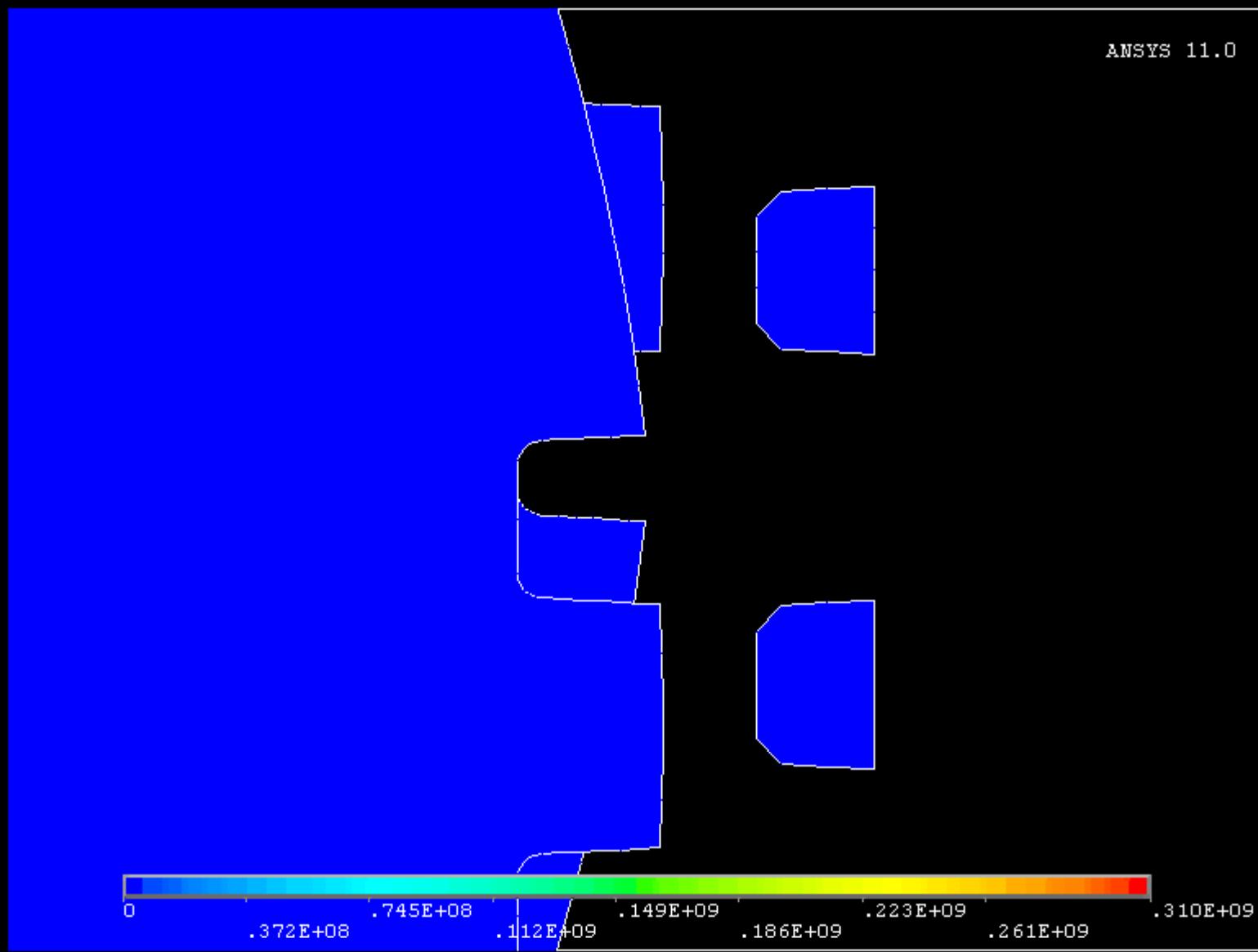


Collaring

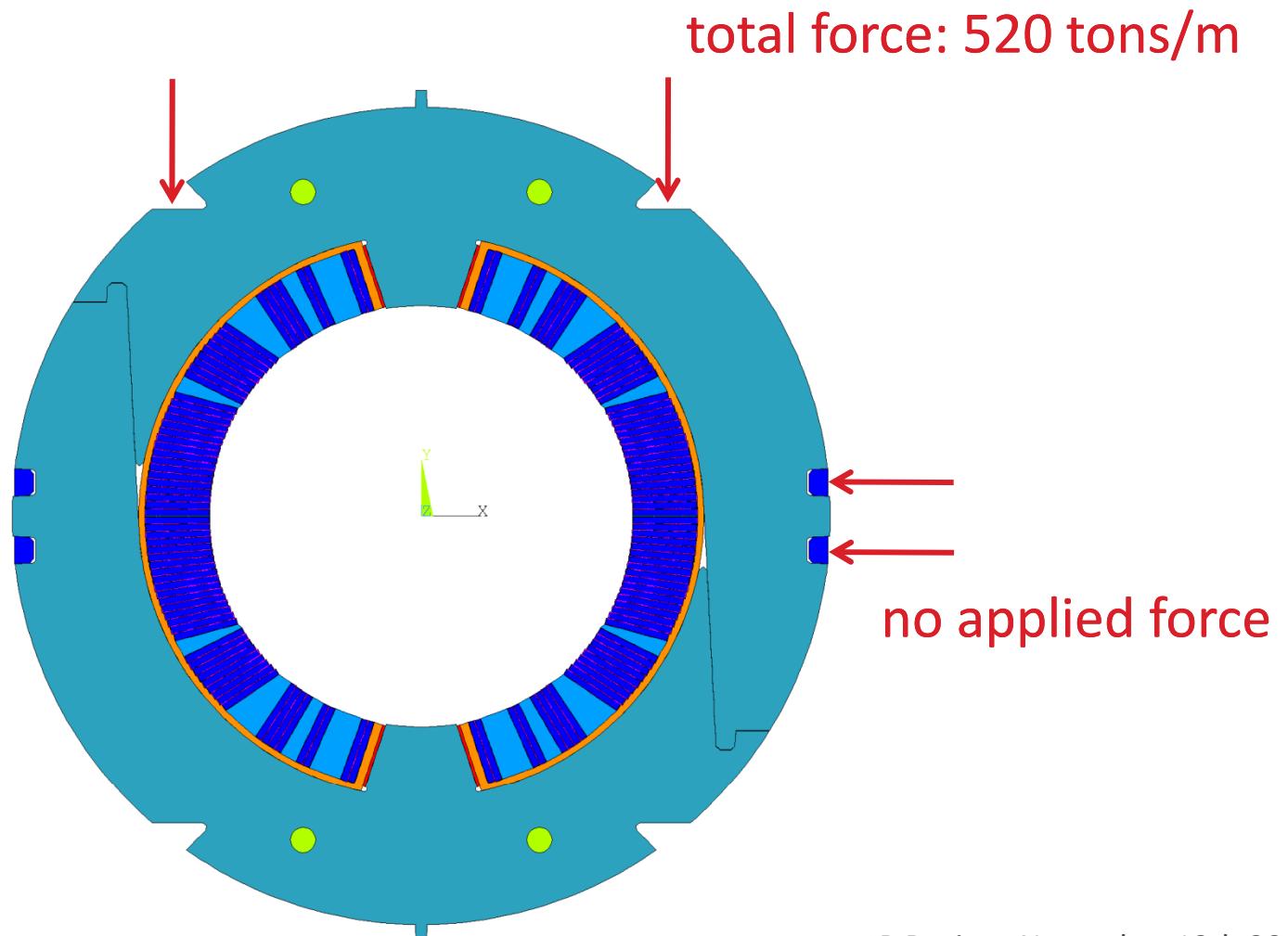
# collars assembly



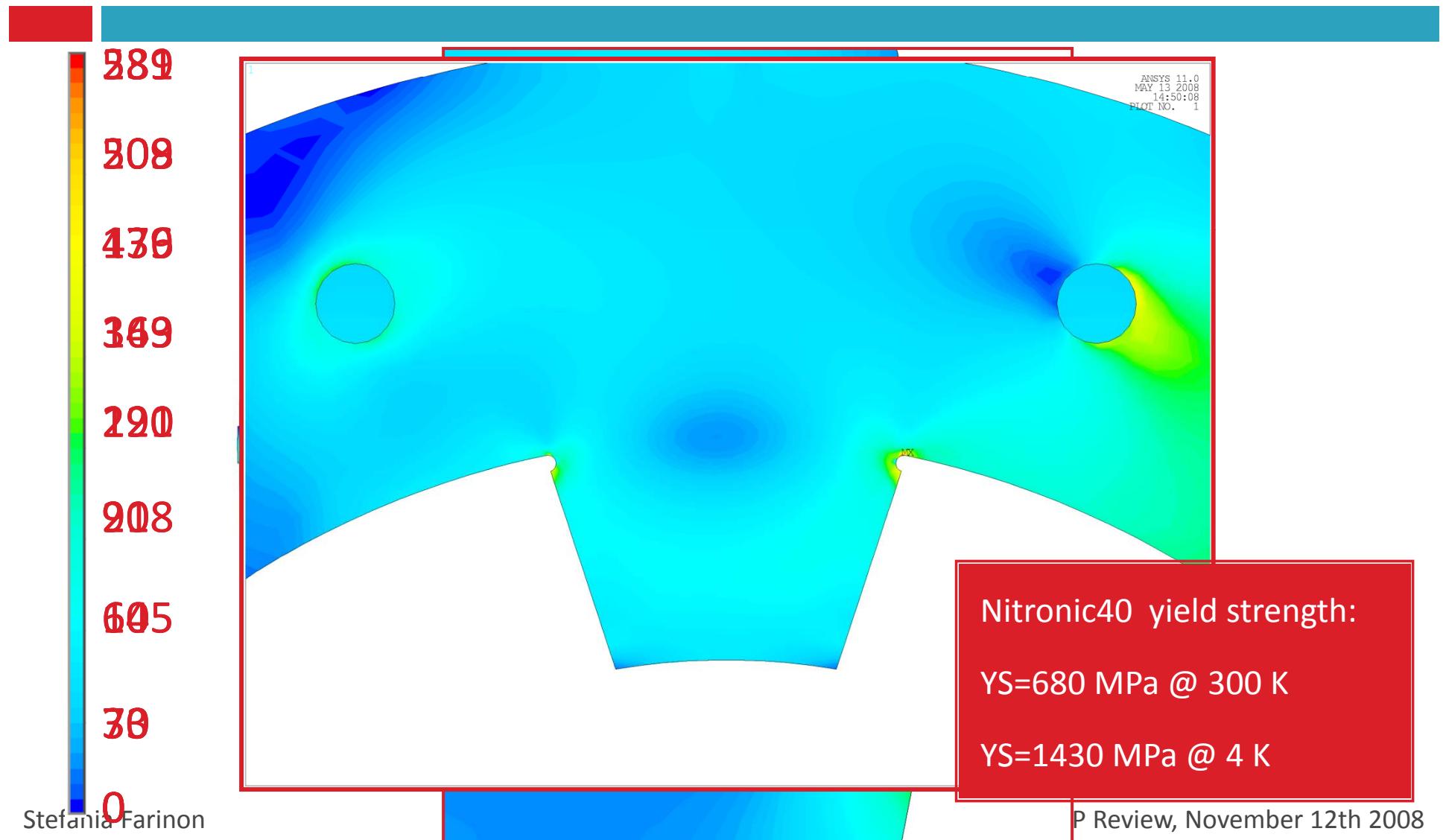
ANSYS 11.0



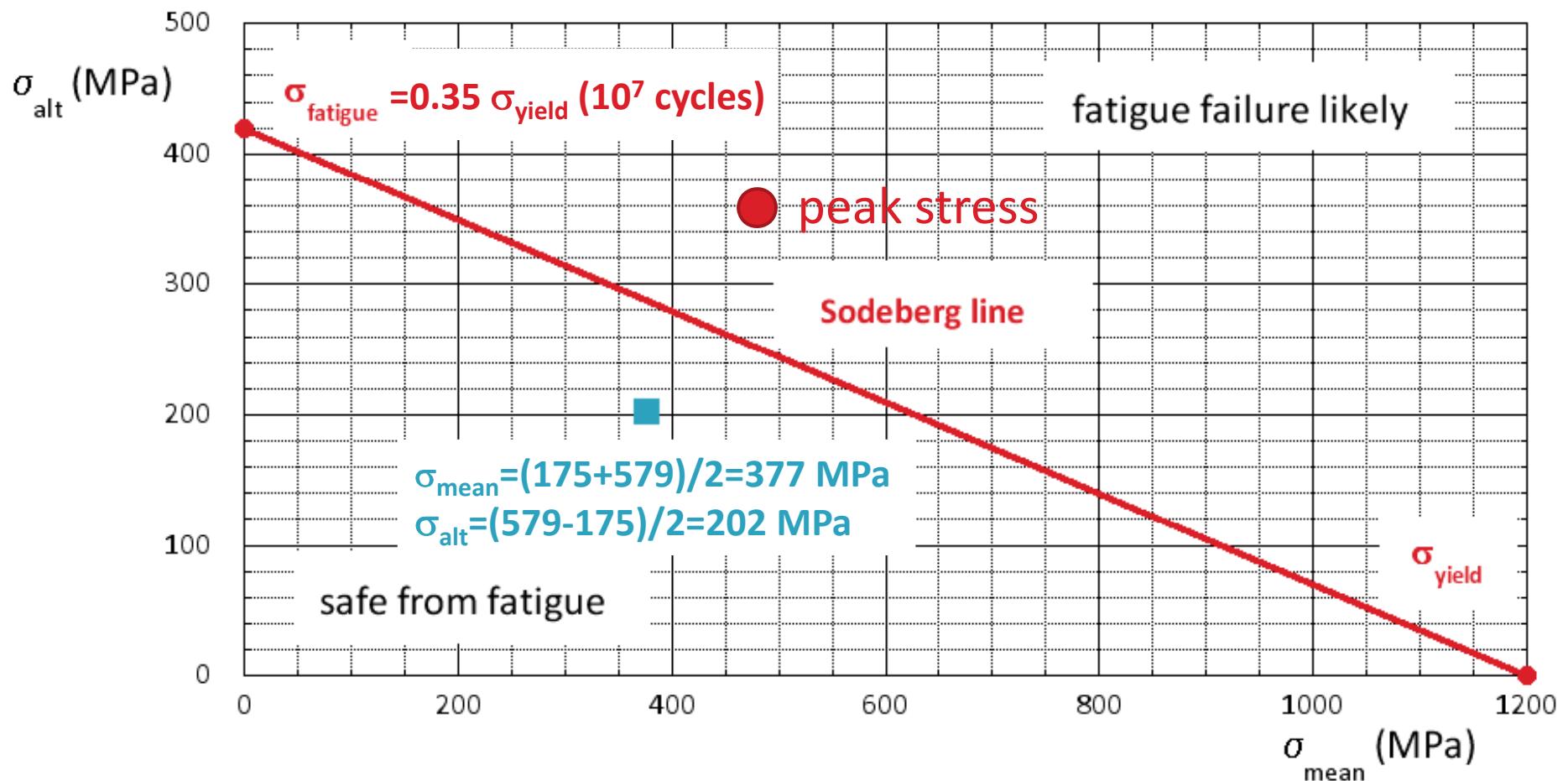
# collaring operation

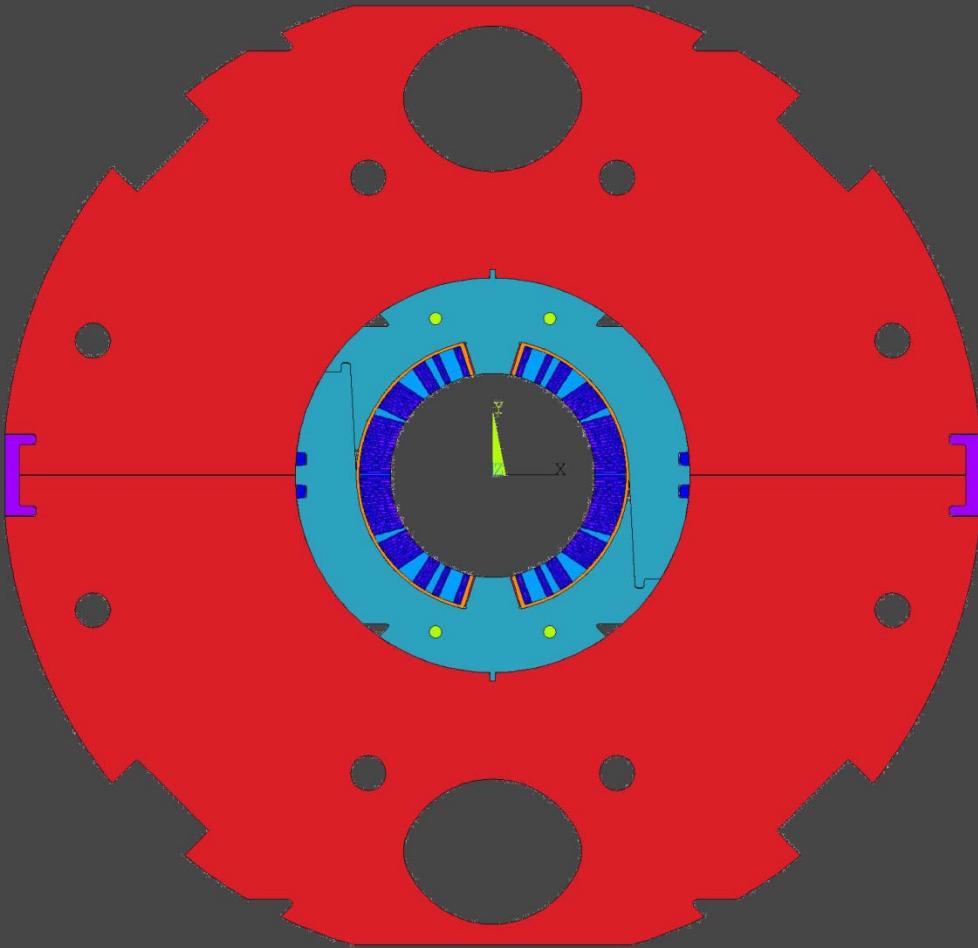


# Von Mises stress after collaring (MPa)



# cool-down and energization of collared-only windings (without iron)





Iron yoke assembly

# iron yoke constraints

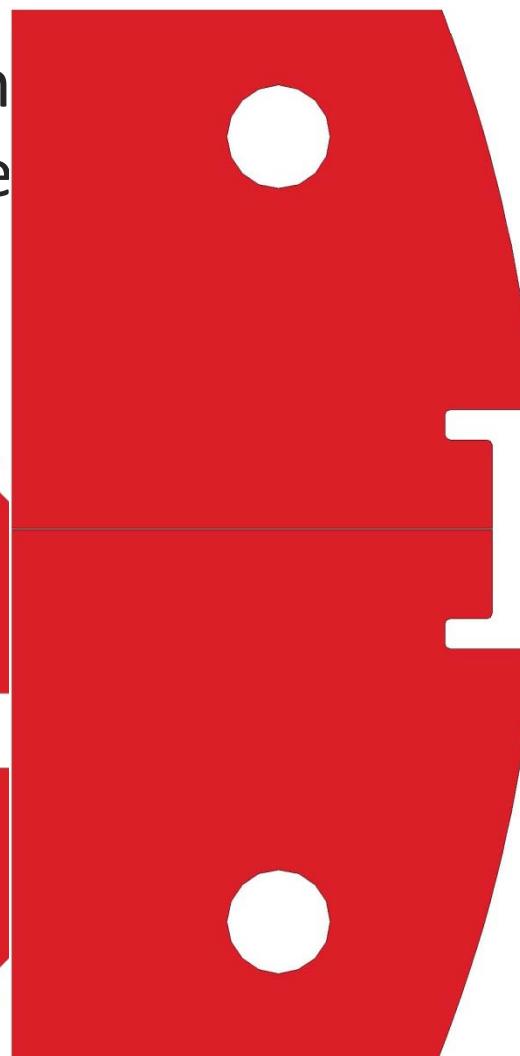
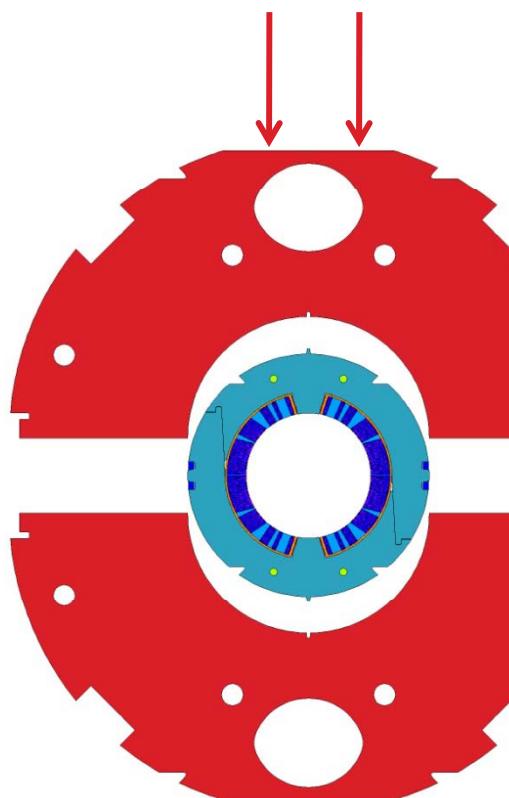
The goal is a side containment to collars.

We should:

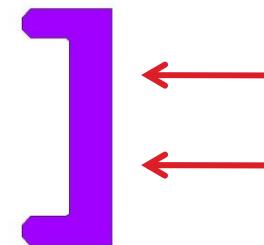
- avoid making a second collaring operation on the winding

# force to close the iron yoke

[ we decided to limit the force to close the iron yoke to 2/3 of the



] close the iron yoke the collars:



C-clamps are heated to be inserted without any applied force

# iron yoke constraints

The goal is a side containment to collars.

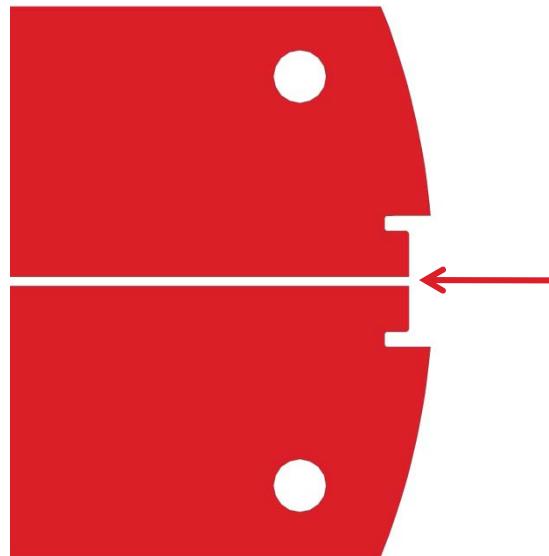
We should:

- avoid making a second collaring operation on the winding
- take into account the effect of the different thermal contractions of stainless steel and iron

# effect of differential thermal contractions

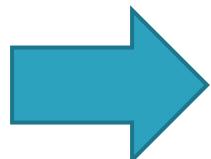
$(\alpha_{\text{iron}} \sim 1.8 \cdot 10^{-3}, \alpha_{\text{Nitronic40}} \sim 2.4 \cdot 10^{-3})$

- we apply a pre-stress to the collared coils through the iron yoke:



80  $\mu\text{m}$  are removed from the midplane

- C-clamps are made of Al-alloy ( $\alpha_{\text{Al alloy}} \sim 4.3 \cdot 10^{-3}$ )



they keep closed the iron yoke

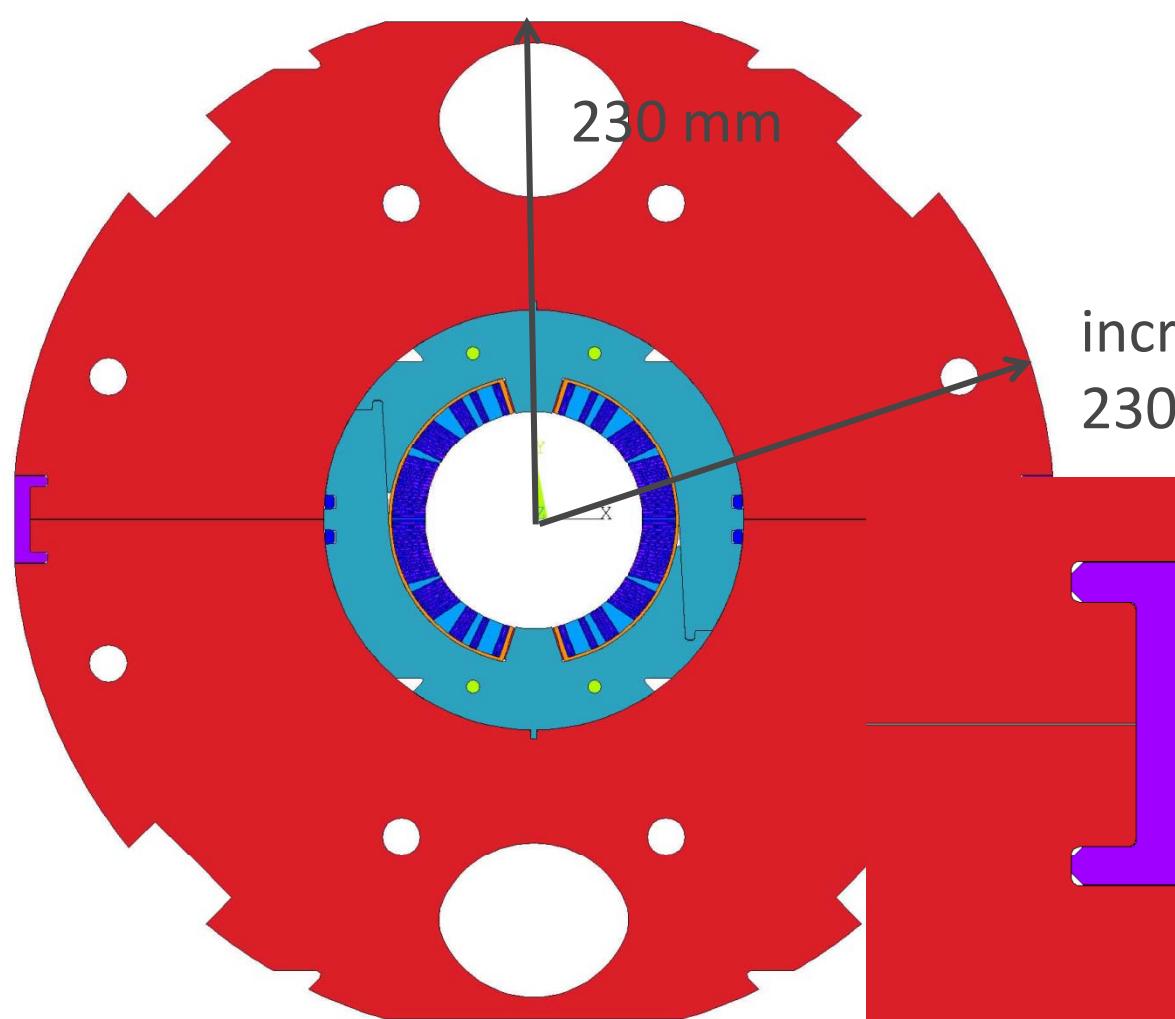
# iron yoke constraints

The goal is a side containment to collars.

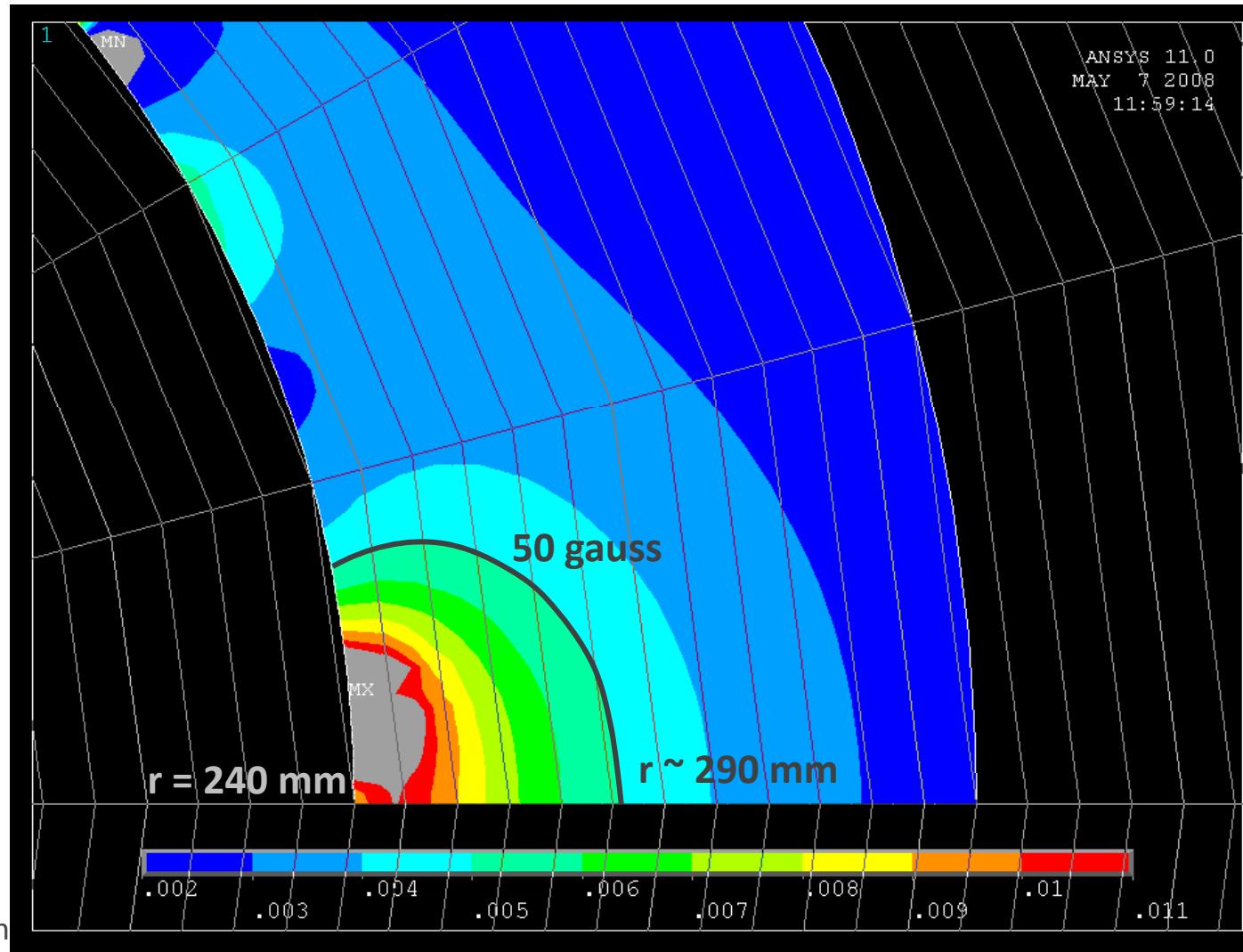
We should:

- avoid making a second collaring operation on the winding
- take into account the effect of the different thermal contraction of iron and stainless steel
- limit the fringe field increase

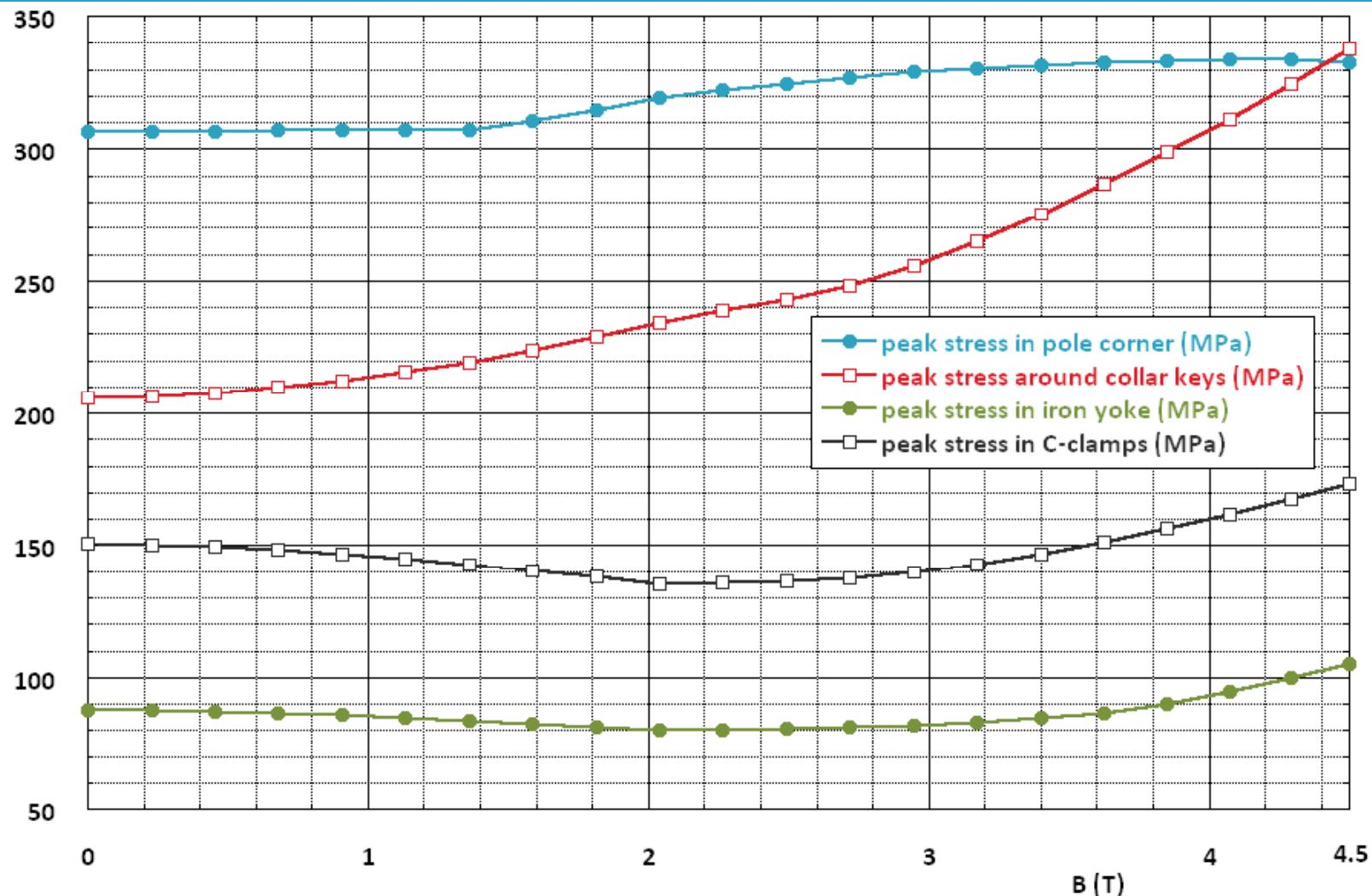
# fringe field



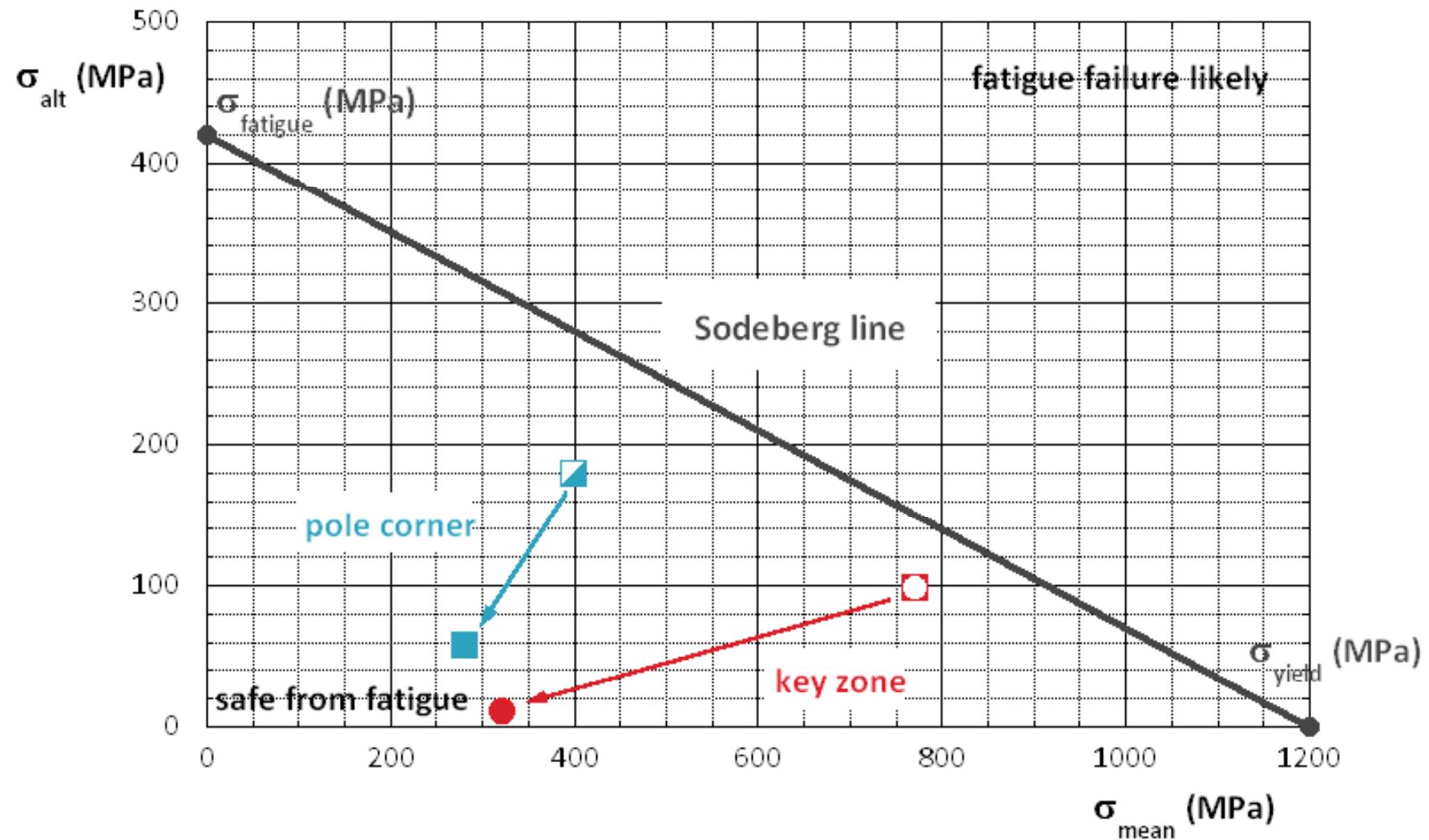
# fringe field



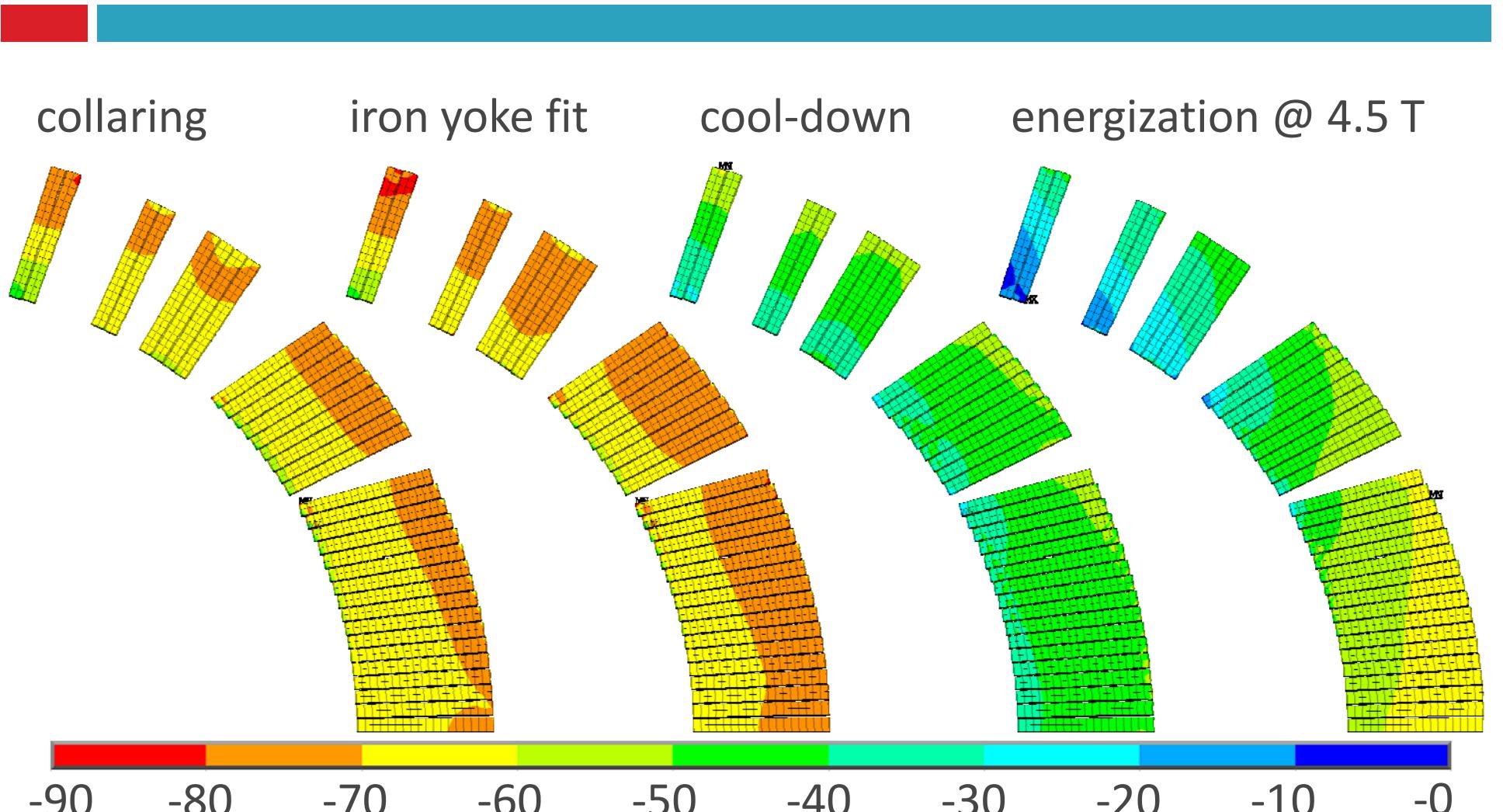
# peak Von Mises stress (MPa)



# fatigue limit in the range 0-4.5 T

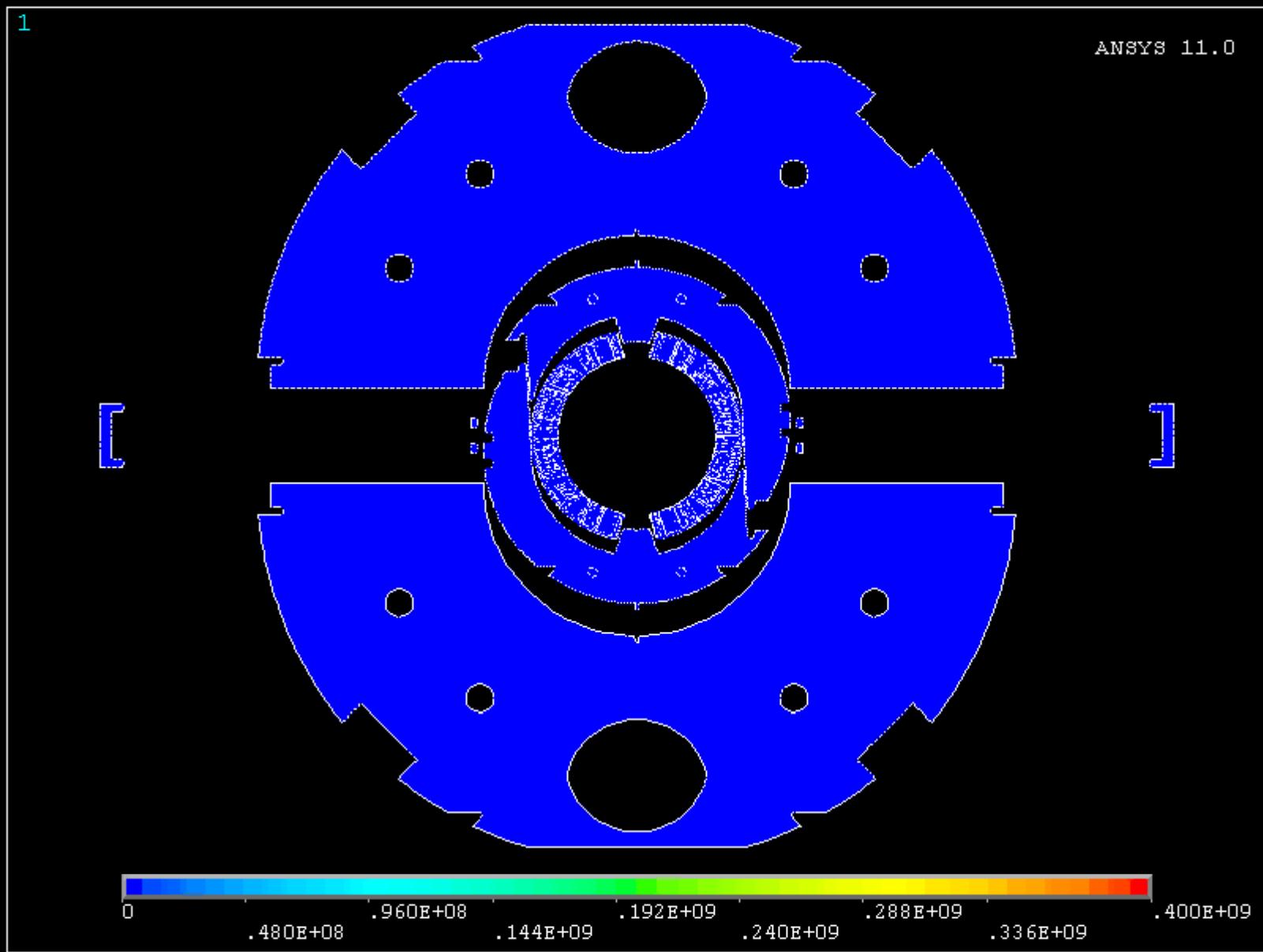


# Azimuthal stress in the winding (MPa)



# Azimuthal stress in the winding (MPa)

	COLLARING	IRON YOKE FIT	COOL-DOWN	ENERGIZATION @ 4.5 T
average azimuthal stress in the winding (MPa)	-68	-70	-44	-48
minimum azimuthal stress of last turn (MPa)	-39	-41	-29	-7

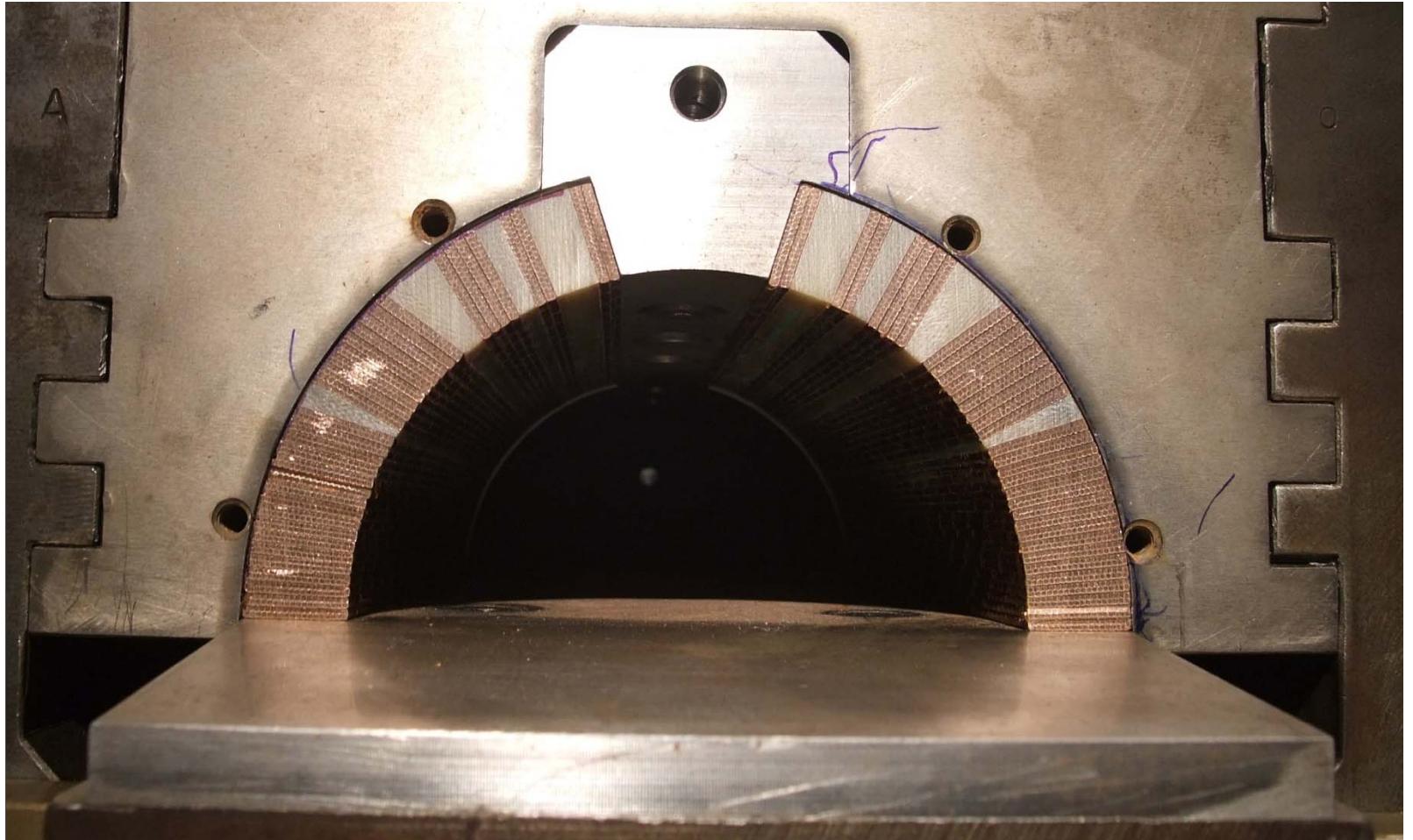


# Conclusions

- The mechanical design requires necessarily both the collars and the iron yoke to limit the stresses
- The peak stresses (MPa) are:

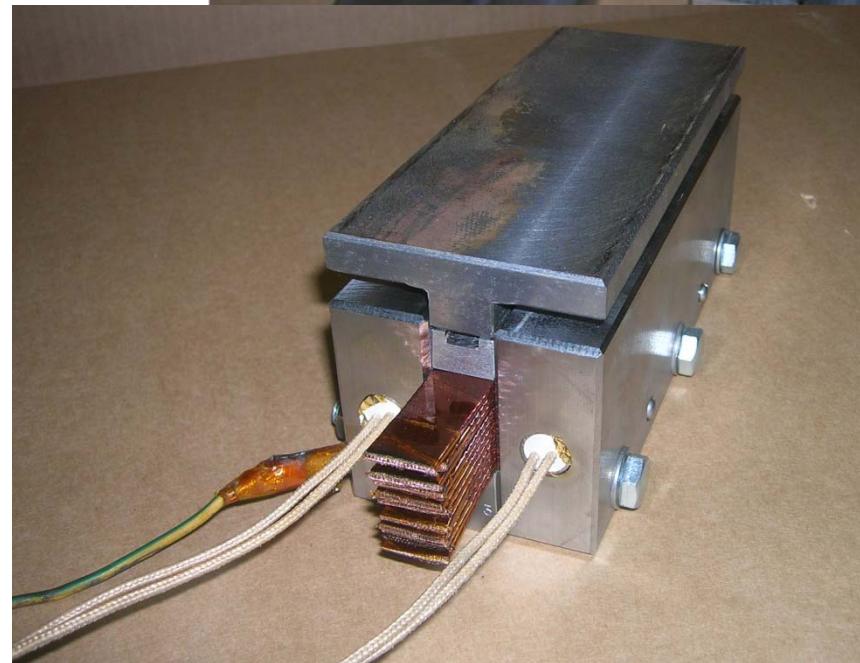
	collars	keys	iron yoke	C-clamps
T=4 K, B=0 T	307	102	92	151
T=4 K, B=4.5 T	338	92	111	175

- The fatigue behaviour is especially critical in this project. The largest stress variation in the collar is 116 MPa, well within the limit of the Sodeberg diagram for 10 millions cycles

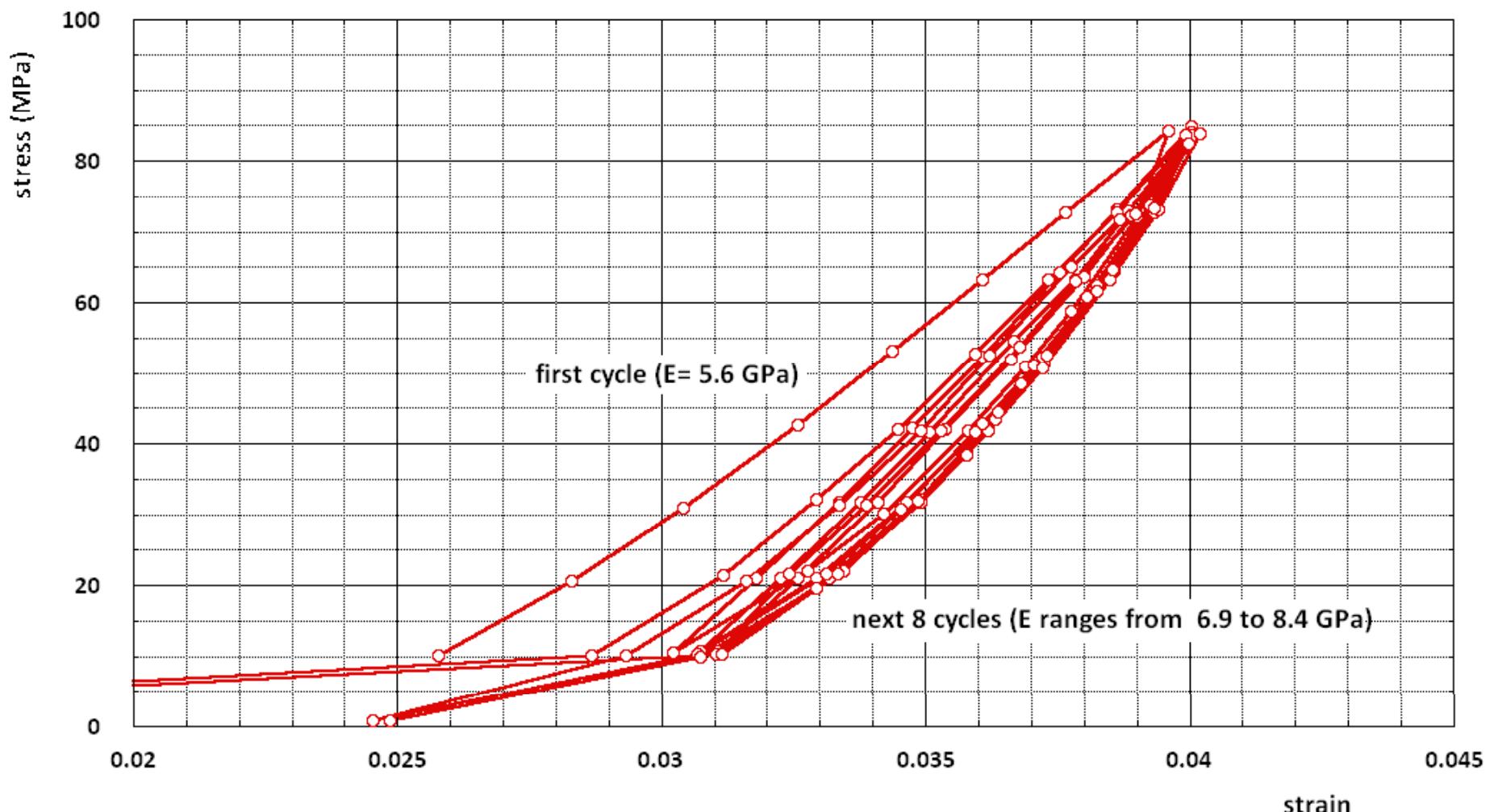


## Mechanical tests

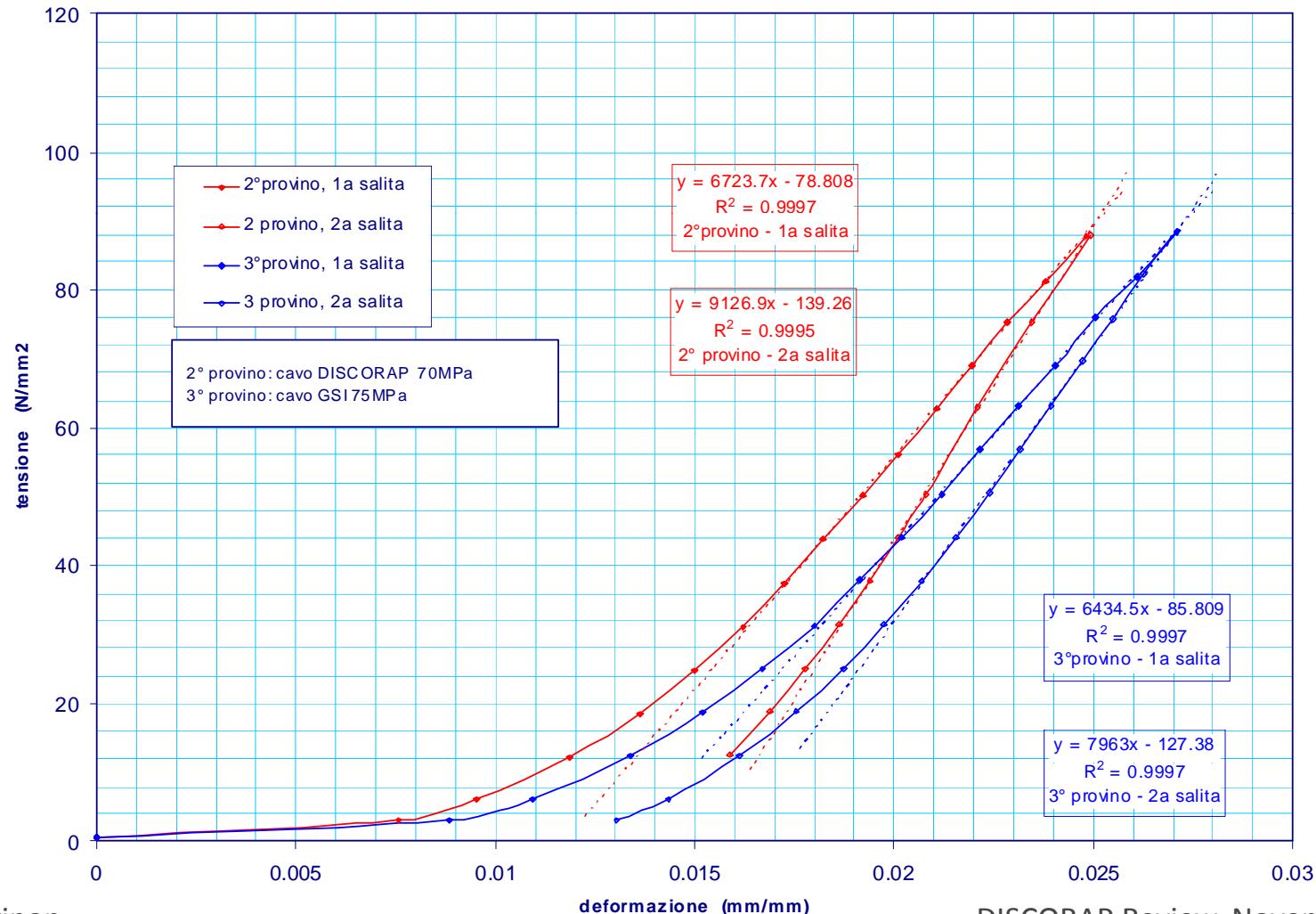
# Stacking tests



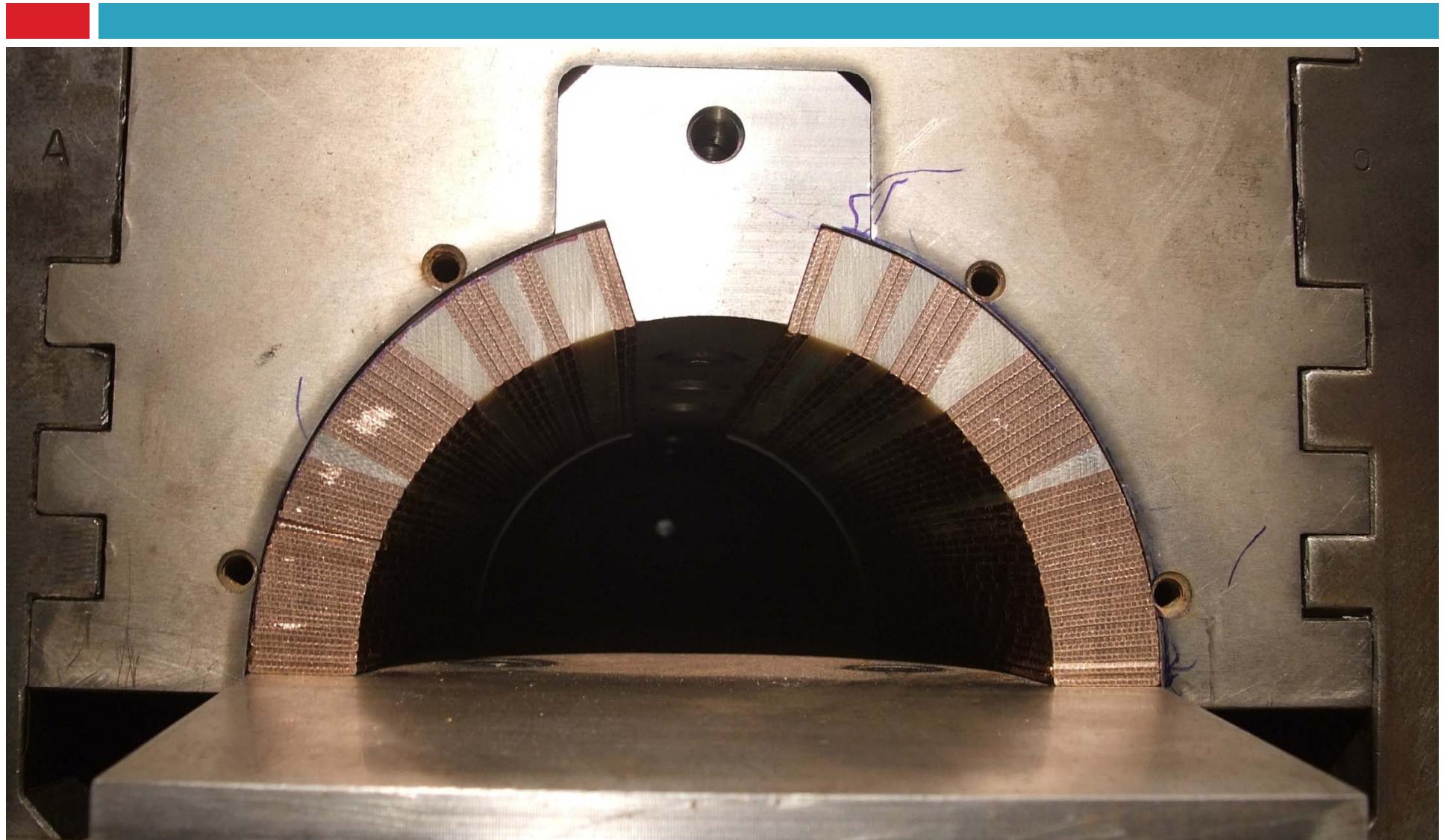
# Typical measurement in ASG



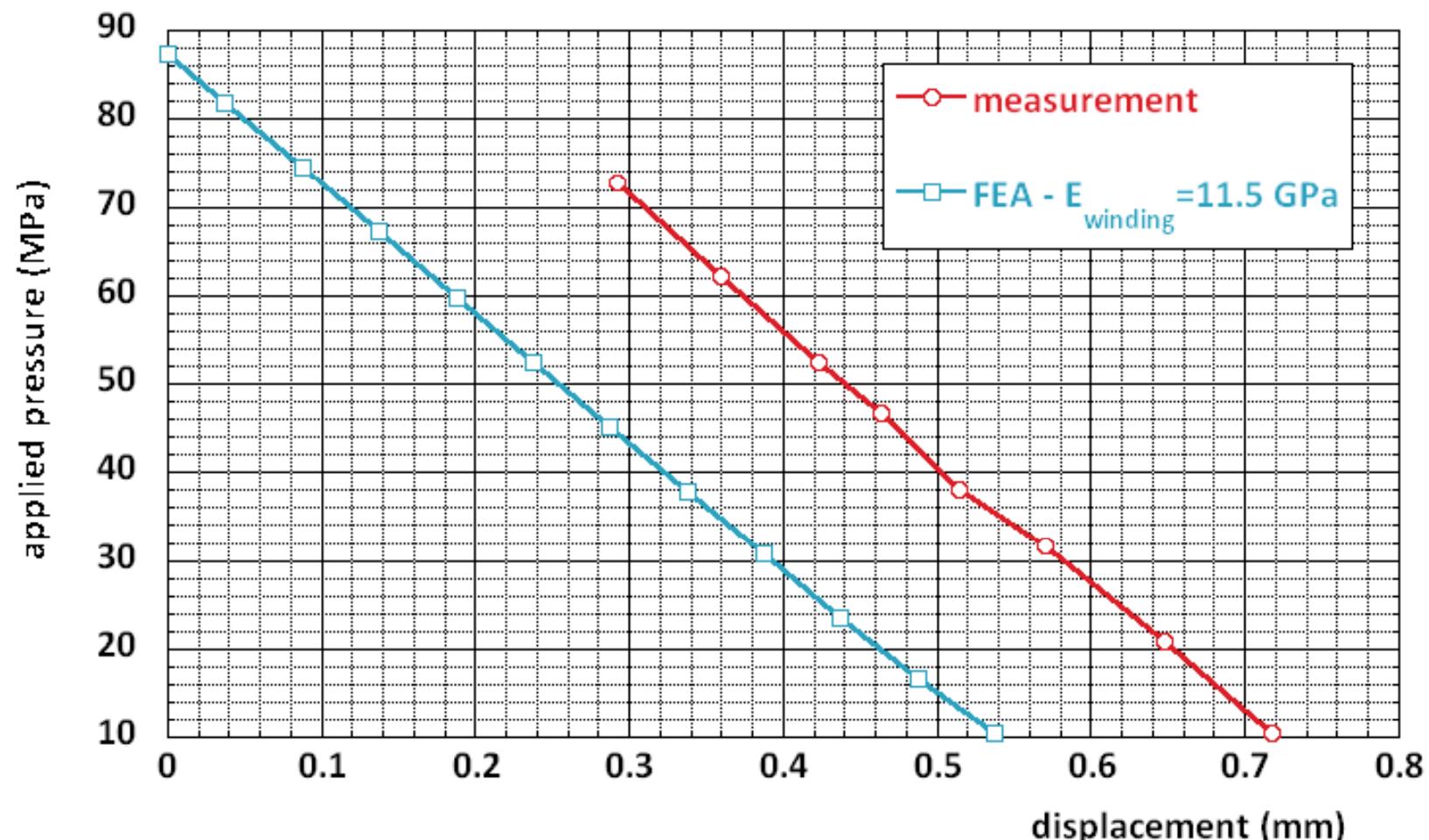
# Typical measurement at LASA



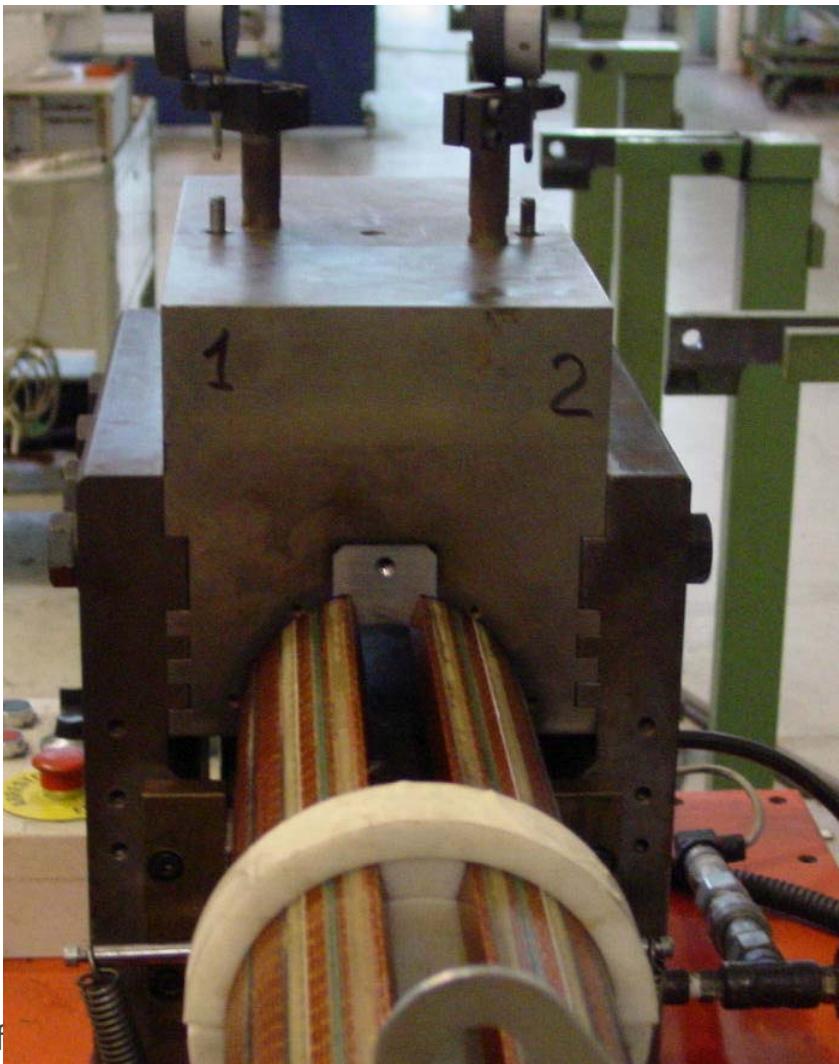
# Arc piled tests



# Arc-piled sample (in the FE model $E=11.5$ GPa)



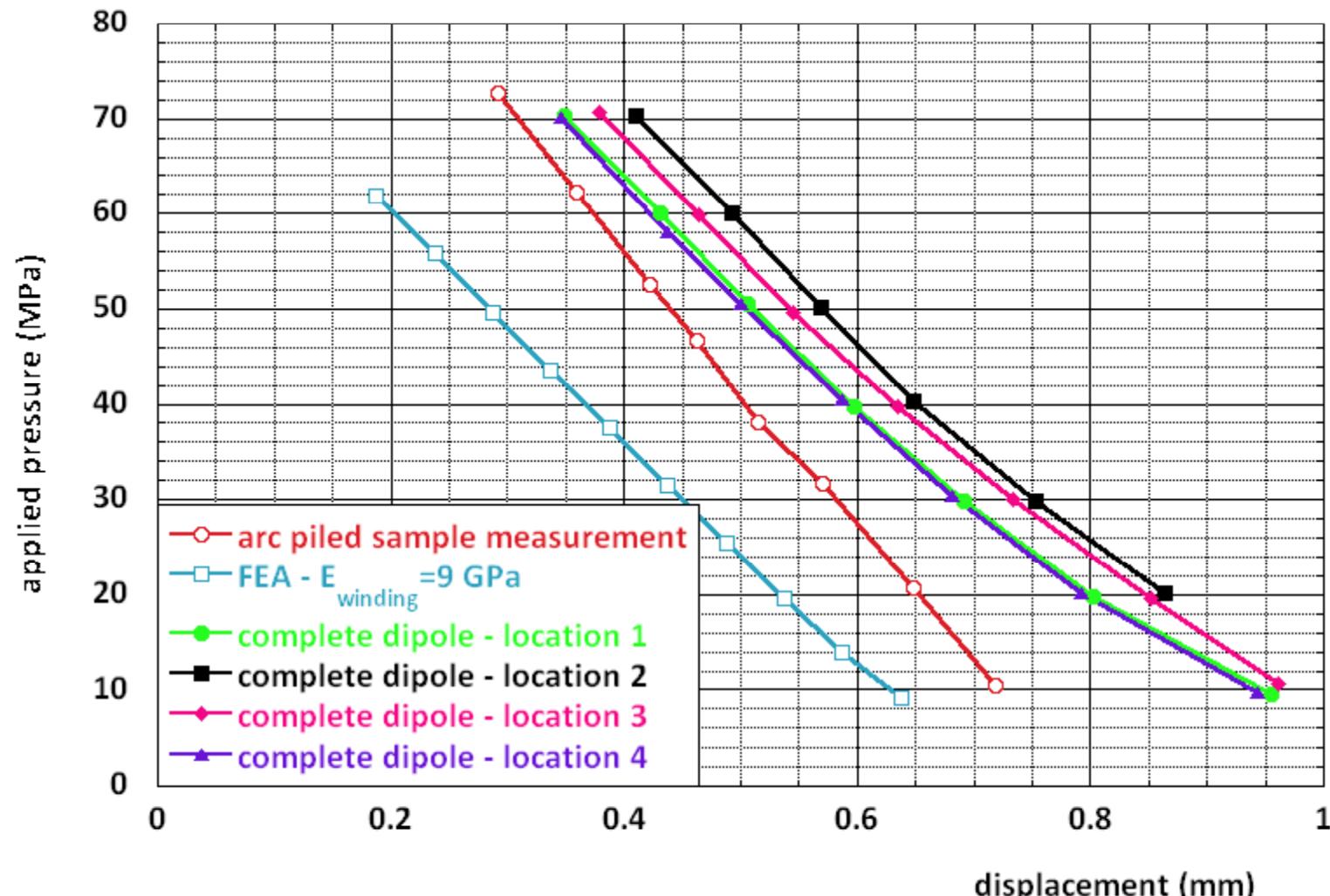
# Complete dipole

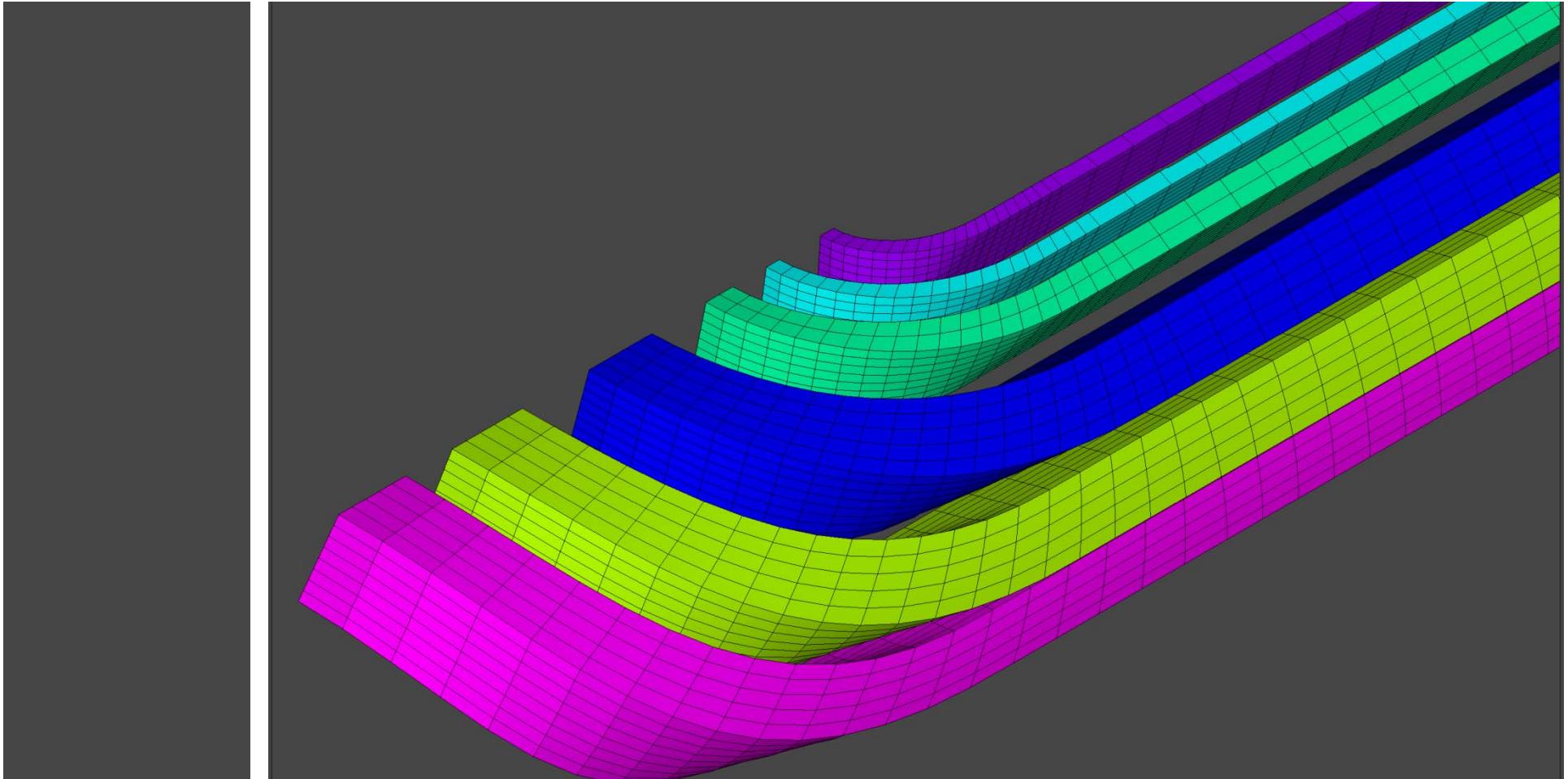


Stef



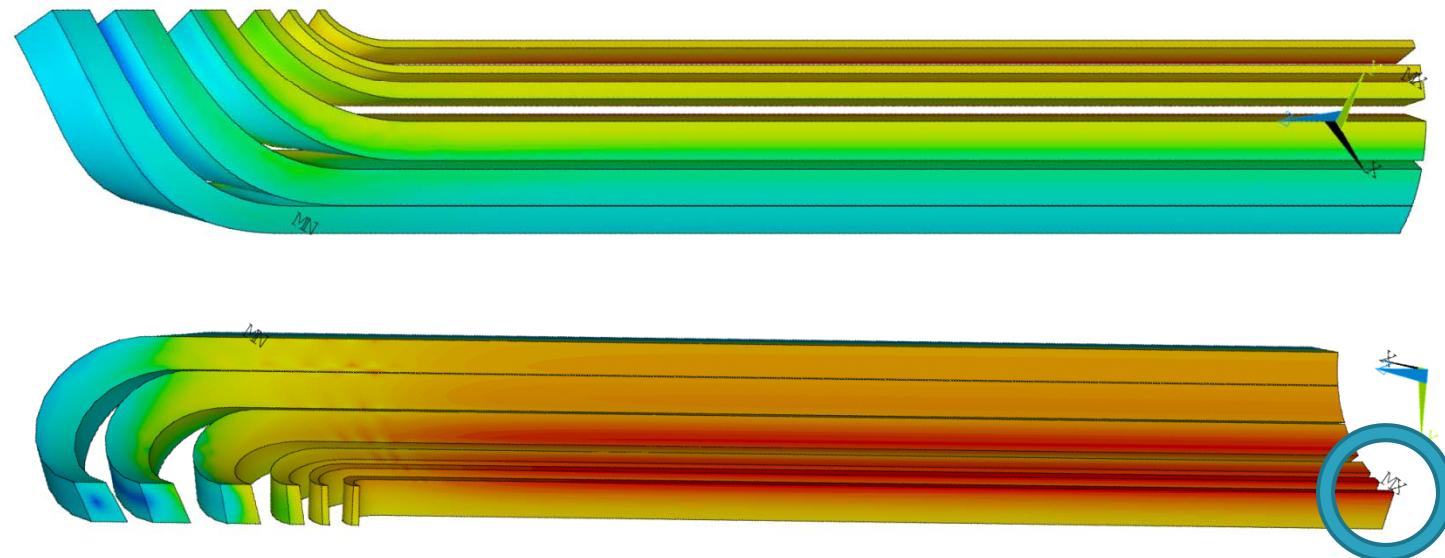
# Complete dipole (in the FE model E=9 GPa)





3D mechanical analysis

# First step: electromagnetic model



ANSYS 11.0  
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15:30:01  
PLOT NO. 1  
.012154  
.550273  
1.088  
1.627  
2.165  
2.703  
3.241  
3.779  
4.317  
4.855

# Lorentz forces

Straight section	Fx (MN/m)	Fy (MN/m)	Fz (MN/m)
3D	+0.678	-0.300	0
2D	+0.684	-0.301	0

Coil ends	Fx (kN)	Fy (kN)	Fz (kN)
3D total	+53.815	-29.324	+23.322
Block #1a	+ 7.2	- 3.6	+ 1.2
Block #1b	+14.0	-11.3	+ 7.2
Block #2	+17.6	-10.1	+ 8.8
Block #3	+ 7.9	- 2.8	+ 3.5
Block #4	+ 3.9	- 1.0	+ 1.6
Block #5	+ 3.1	- 0.5	+ 1.0

# Second step: mechanical analysis

